

## SUBSTITUTE SPECIFICATION

### FLEXIBLE MICROCHANNEL HEAT EXCHANGER

#### STATEMENT OF GOVERNMENT INTEREST

This invention was made with government support under Contract Number DABT63-97-C-0069 awarded by the Defense Advanced Research Project Agency (DARPA). The government has certain rights in this invention.

#### FIELD OF THE INVENTION

A field of the invention is heating and cooling. An additional field of the invention is mesoscopic devices.

#### BACKGROUND OF THE INVENTION

Small scale active heating and cooling devices hold tremendous potential. Potential uses are limited only by the decision as to whether a device, process, or application would benefit from active heating or cooling. Implementation of networked, low-power mesoscopic devices offers obvious advantages compared to traditional active heating and cooling. Practical issues remain in the way of widespread implementation and use of such devices, however. In addition to active heating and cooling devices, e.g., heat pumps, there are additional examples of mesoscale systems that hold promise for a wide range of practical applications. Examples of such mesoscale systems include combustors and evaporators, heat exchangers, and chemical and biological systems.

Mesoscale devices such as these can be defined as ones where the critical physical length scale is on the same order as the governing phenomenological length scale, or ones with critical dimensions that span the microscale to the normal scale ( $\mu\text{m} < \text{length scale} < \text{cm}$ ). These large differences in scale pose several challenges in manufacturing. Mesoscopic heat exchangers are needed for a number of applications requiring high heat flux ( $> 1000 \text{ W/m}^2$ )

1 across thin cross-sections, without incurring excessive pressure losses due to fluid  
2 flow in small channels. Enhancement in heat transfer occurs when the effective  
3 cross-sectional thickness of a mesoscale heat exchanger matches the thickness  
4 over which heat is transferred to the working fluids.

5 Exemplary potential practical uses of heat exchangers include laptop  
6 computer cooling, car seat heating and cooling, airfoil skin heat exchangers,  
7 micro-chemical reactors, and compact heat exchangers among others. Another  
8 exemplary practical application is the temperature control of clothing. While time  
9 is likely to bring the technology to clothing in general, a likely initial application is  
10 to chemical and biological warfare protective suits for military personnel operating  
11 in extremely hazardous environments. Integrated mesoscopic cooler circuits  
12 (IMCC) have been developed by some of the present inventors, and are described,  
13 for example in Beebe et al., U.S. Patent 6,148,635, which is incorporated by  
14 reference herein. Also see, Shannon, et al., "Integrated Mesoscopic Cooler  
15 Circuits (IMCCs)." Proceedings of the ASME, Advanced Energy System Division  
16 39, Symposium on Miniature and Mesoscopic Energy Conversion Devices (1999),  
17 p. 75-82.

18 Others have endeavored to design, fabricate, and mass-produce  
19 microchannel (below about 1mm diameter) heat exchangers for microelectronics  
20 cooling and the refrigeration industry. See, P.M. Martin et al, "Microchannel Heat  
21 Exchangers for Advanced Climate Control," Proceedings of the SPIE 2639,  
22 (1995), p. 82-88. Delphi Automotive Systems and Modine Manufacturing  
23 Company have produced some commercially available mesoscopic heat  
24 exchangers made from extruded metals, such as aluminum. Such exchangers are  
25 capable of holding high internal pressures and can support large heat fluxes, but  
26 typically measure between 0.5 to 1 mm thick, and are not flexible after forming.

27 Microfabricated thin-film heat exchangers with microchannels 1 mm  
28 wide x 30  $\mu\text{m}$  high, made from photosensitive polyimide layers have been  
29 reported. Mangriotis, M. D. et al., "Flexible Microfluidic Polyimide Channels,"

1 Transducers 99, The 10th International Conference on Solid-State Sensors and  
2 Actuators, Digest of Technical Papers, Sendai, Japan, June 7-10, (1999) p. 772-  
3 775. Polyimide was chosen because it is a commercially available high-  
4 performance polymer, renowned for its excellent thermal stability, mechanical  
5 toughness, high strength, and superior chemical resistance. Fabrication of these  
6 heat exchangers utilized batch-mode semiconductor processing of multiple spin-  
7 coated layers of DuPont (now HD Microsystems) PI-2721 polyimide to define  
8 specific fluid and vent channel geometries, followed by solvent bonding of a 75  
9 mm thick Kapton HN film to seal the device. See, Glasgow, I. K. et al., "Design  
10 Rules for Polyimide Solvent Bonding," Sensors and Materials 11.5 (1999) p. 269-  
11 278.

12 Even with properly designed vent channel spacing, vapor evolution  
13 inherent to the solvent bonding technique can locally degrade the interfacial seal  
14 between the microchannels and the Kapton HN film. Thus, large area heat  
15 exchangers demonstrated poor structural reliability and thus low fabrication yields.  
16 Sealed devices inevitably suffered from very high pressure losses ( $> 100$  kPa) over  
17 flow lengths of 20 mm, caused by the 30 micron interior channel height. To  
18 minimize pressure losses over long flow paths, increased channel heights are  
19 required. However, achieving 50 to 150  $\mu\text{m}$  high channels by using multiple spin-  
20 coated layers proved to be difficult to scale-up over large planar areas. These  
21 examples illustrate some of the difficulties faced in mesoscale device fabrication.  
22 Mesoscale devices with vastly different critical dimensions require fabrication  
23 methods that can simultaneously meet the tolerances required at both scales.

## 24 SUMMARY OF THE INVENTION

25 A flexible mesoscopic heat exchanger is provided by the invention.  
26 The heat exchanger of the invention includes uniform microchannels for fluid  
27 flow. Separate header and channel layers include microchannels for fluid flow  
28 and heat exchange. A layered structure with channels aligned in multiple

1 orientations in the layers permits the use of a flexible material without channel  
2 sagging and provides for uniform fluid flows. In a preferred embodiment, layers  
3 are heat sealed, e.g., by a preferred lamination fabrication process.

#### 4 BRIEF DESCRIPTION OF THE DRAWINGS

5 FIG. 1 is an exploded schematic view of a preferred embodiment  
6 mesoscopic heat exchanger;

7 FIG. 2 is a schematic assembled view of the preferred embodiment  
8 mesoscopic heat exchanger;

9 FIG. 3 is a block diagram illustrating a preferred fabrication process  
10 for a mesoscopic heat exchanger; and

11 FIG. 4 shows the time, temperature, and applied pressure profile  
12 found to optimally bond layers in a laboratory conditions and style fabrication of a  
13 mesoscopic heat exchanger.

#### 14 DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

15 The invention concerns a mesoscopic multilayer structure with  
16 internal microchannels. The entire structure is flexible. A layered structure with  
17 channels aligned in multiple orientations in the layers permits the use of a flexible  
18 material without channel sagging. Flows are through separate manifold and  
19 channel layers. A fabrication method of the invention includes single layer  
20 patterning and multilayer lamination. Heat bonding avoids solvent bonding.

21 Referring now to FIG. 1, a preferred embodiment heat exchanger  
22 includes layers 22a, 22b, 22c and 22d. Each of these layers is formed of flexible  
23 heat-sealable polyimide. Layers 22b and 22c include uniformly dimensioned (in  
24 width and height) microchannels 24. From device to device, dimensions of the  
25 channels may be selected to meet a particular performance parameters, but within  
26 each individual device, microchannels are highly uniform in width and height.  
27 Refrigerant or other fluid enters through an inlet hole 26 the device interface in  
28

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1 layer 22d. The device interface layer 22d interfaces with another device that  
2 includes means for promoting flow of liquid through the heat exchanger. Layer  
3 22c acts as a header, i.e., a layer for even distribution of refrigerant or heating  
4 fluid for heat transfer into the channel layer 22b. Heat transfer is with the cap  
5 layer 22a that seals in refrigerant by closing the top of channels 24 in the channel  
6 layer 22b and forms an outside surface of the heat exchanger. An opposite side of  
7 the header layer reaccepts refrigerant after heat transfer and creates a uniform flow  
8 back into an exit hole 28 of the device interface layer 22d.

9           The microchannels 24 in alternate layers, e.g., layers 22b and 22c are  
10 oriented differently to provide channel floors (the individual layers 22b and 22c  
11 only define, by themselves, channel walls), and add a structural integrity that  
12 avoids sagging of thin-walled and thin-floored microchannels in the completed  
13 assembly. In addition, the lengths of individual microchannels are patterned in a  
14 manner to establish uniform flows. In the preferred FIGs. 1 and 2 embodiment,  
15 for example, microchannels in layer 22b have different lengths that establish a  
16 shape. The center channels are gradually shorter to give the channels in the layer  
17 an overall hourglass like configuration. The waist 31 of the hourglass shape  
18 avoids channels over ports 30 in the layer 22c that communicate refrigerant into its  
19 channels from the inlet hole 26 and out from its channels into the outlet hole 28.  
20 In intersection areas 32 (see FIG. 2) where channels from the layers 22b and 22c  
21 overlap, the different orientation provides rigidity that avoids channel sag under  
22 pressured conditions. Only a few of the many intersections 32 in FIG. 2 are  
23 labeled with reference numerals to keep the figure clear. Referring to FIG. 2, the  
24 shape also establishes the desirable uniform flows into channels. Uniform flows  
25 into and out of the exchanger avoid pockets of pressure build-up that can be  
26 destructive to the heat exchanger.

27           When manifold input area from ports 30 to each channel in the layer  
28 22b is varied, with channels closest to the ports 30 having a minimum area and  
29 channels farthest from the ports 30 having a maximum area, refrigerant flow is

1 optimized. The general star-burst manifold shape surrounding ports 30 is, along  
2 with the hourglass configuration in the channel layer 22b, therefore preferred to  
3 provide uniform flows. A set 36 of microchannels in the channel layer 22b furthest  
4 from the ports 30 intersects all of the microchannels in the header layer 22c,  
5 whereas the number of header microchannels intersected by microchannels in the  
6 channel layer 22b gradually decreases (by sets in the preferred channel layer 22b)  
7 with a set 38 of microchannels closest to the ports intersecting the fewest number  
8 of microchannels in the header layer 22c. The number of cross-over intersections  
9 32 between the channels in header layer 22c and channel layer 22b controls the  
10 input area afforded each flow into a set of the microchannels in the channel layer  
11 22b.

12               An additional point about the shaping is that the patterns make use  
13 of separate header flow layer 22c to enable fabrication by a lamination process.  
14 From a fabrication standpoint, the lamination process can only be utilized if each  
15 individually patterned layer represents a contiguous whole, with no independent or  
16 isolated solid geometries. Overlapping of geometrical material voids patterned in  
17 the individual layers during the lamination process creates a manufacturable  
18 internal geometry and defines channels when the individual layers 22b and 22c  
19 have a piano-wire style cut all the way through to define channel walls. This is  
20 achieved by the separate header 22c and channel 22b layers, resulting in three-  
21 dimensional, rather than two-dimensional, refrigerant flow paths.

22               In accordance with the preferred embodiment, layers 22a, 22b, 22c  
23 and 22d are formed from heat-sealable polyimide films. Lamination of a  
24 multilayer structure of mechanically patterned polyimide heat-sealable films was  
25 found to provide the most versatile fabrication process. It is critical to use heat  
26 sealed films, as contrasted with solvent bonded films. Exemplary heat-sealable  
27 polyimide films preferred for the invention are the Kapton® KJ and EKJ (DuPont)  
28 films. Other examples are Teflon® coated Kapton® FN heat-sealable films.  
29 Other heat-sealable polyimide films, including those to be developed, will also be

1 suitable. In contrast to Teflon® coated Kapton® FN heat-sealable films, Kapton®  
 2 KJ and EKJ (DuPont) are thermoplastic all-polyimide films designed as adhesive  
 3 bonding sheets for high performance applications. The difference between KJ and  
 4 EKJ films is the inclusion of a Kapton® E polyimide layer as the core of an EKJ  
 5 film to enhance its mechanical properties. The enhanced properties are preferred.

6 The EKJ films for the cap 22a and inlet/outlet 22d layers prevented,  
 7 due to their higher modulus and glass transition temperature, sagging of the  
 8 spanning membrane sections of the microchannels and manifolds during the  
 9 lamination cycle. Omission of the EKJ layers in attempts to use KJ for all four  
 10 layers resulted in solid laminates with no internal geometry because of  
 11 thermoplastic flow during the bonding process. Accordingly, heat sealable  
 12 polyimide layers used for the outer layers must have a sufficiently high modulus  
 13 and glass transition temperature to maintain solidity during the lamination process.  
 14 Table 1 highlights a few selected properties of the preferred materials:

15 TABLE 1

	KJ	EKJ
<b>Glass Transition Temperature</b>	220°C	220°C KJ >340°C E core
<b>Tensile Strength</b>	20 ksi	30 ksi
<b>Modulus</b>	400 ksi	700 ksi
<b>Elongation</b>	150%	70%
<b>CTE</b>	60 ppm/°C	25 ppm/°C
<b>Moisture Content</b>	1.0%	2.0%

1 Channel and manifold heights are easily controlled by layer  
2 thickness. With single channel layer construction, microchannel heights of  
3 roughly 70  $\mu\text{m}$  were achieved in experimental prototypes according to the FIGs.1  
4 and 2 embodiment.

5 Referring now to FIG. 3, a block diagram illustrates the general steps  
6 for a preferred fabrication method of the invention. Heat-sealable polyimide  
7 sheets are cut to size (step 34). Mechanical patterning of the layers is conducted  
8 (step 36). A preferred technique is computer controlled knife cutting for the  
9 mechanical patterning. In practice, there are likely four process flows, one for  
10 each of the four layers 22a, 22b, 22c, 22d. Subsequent to patterning, the layers  
11 undergo bond preparation (step 38), e.g., solvent degreasing and a dehydration  
12 bake. Layers are aligned (step 40) and laminated (step 42) by a heat treatment,  
13 such as a vacuum hot press.

14 In a preferred technique for the mechanical patterning of step 36  
15 used to form experimental prototype heat exchanges, layers were patterned using  
16 computer controlled knife cutting. In prototypes constructed according to the  
17 preferred FIGs. 1 and 2 embodiment, layers 22a and 22d were made from EKJ  
18 (50 $\mu\text{m}$  thick) films, and layers 22b and 22c were made from KJ (75 $\mu\text{m}$  thick)  
19 films. In practice of the invention, thicker films for layers 22b and 22c would be  
20 preferred to allow deeper microfluidic channels.

21 To begin the preferred patterning process, sheets of KJ and EKJ are  
22 sheet cut (step 34) into roughly 400 mm x 400 mm areas. The patterning used a  
23 mounting (step 44) onto a carrier. In the experimental fabrication, paper-board  
24 with an adhesive backing was used as a carrier for the polyimide films during the  
25 patterning process. The depth of cut was set to approximately 80  $\mu\text{m}$  so that the  
26 blade does not penetrate the paper-board carrier, ensuring that sectioned film areas  
27 remain attached to the carrier and do not project outward and interfere with the  
28 traveling blade. After initial manual alignment, the sheet is positioned into the



1   grit-rolling cutting plotter (step 46) that automatically provides horizontal and  
2   vertical justification. Cutting proceeds according to a 3 dimensional modeling  
3   (step 48). A three-dimensional solid model controls the cutting process (step 50).  
4   The carrier is removed after cutting (step 52). With the use of a paper carrier, the  
5   carrier board may be removed, for example, by soaking in an acetone bath for a  
6   time to permit the acetone to diffuse through the paper board to the  
7   adhesive/polyimide interface, dissolving the adhesive backing. The patterned  
8   polyimide films “lift-off” the paper board. No peeling or stretching of the films is  
9   required for removing the carrier substrate, precluding any unwarranted straining  
10  of the individual layers and patterns.

11               The completed cutting process contaminates the polyimide layers.  
12  The bond preparation step 38 prepares the layers for lamination. Contaminated  
13  layers may not bond properly. A second acetone bath may be used for solvent  
14  degreasing (step 54). During the degreasing (step 54), mechanical scrubbing (step  
15  56) may be used, e.g., with polyester-fiber cloths, to remove residual adhesive as  
16  well as other organic contaminants present on the film as received from the  
17  factory. Layers are rinsed (step 58), e.g., with an isopropanol bath, and blown dry  
18  (step 60), e.g., with nitrogen. After bond preparation, films should be handled with  
19  sterile equipment or, if by operators, with operators wearing powder-free latex or  
20  nitrile gloves. Surface cleanliness tends to dominate the mechanical and chemical  
21  strength of interlaminar bonds.

22               Test fabrications of prototype heat exchangers revealed that KJ and  
23  EKJ films, like most all polyimides, demonstrated a propensity to absorb water in  
24  ambient temperature and humidity environments. During the high-temperature  
25  lamination process, absorbed water volatilized, aggregated, and formed voids at the  
26  layer interfaces, making it extremely difficult to bond large areas. Void formation  
27  is avoided by a vacuum dehydration bake (step 62) prior to lamination. In  
28  experiments, a 12 hour bake at a temperature of 150°C and an ambient pressure of  
29  0.1KPa was used. The dehydration bake time and temperature schedule was not

1 optimized, and thus shorter process times are thought to be possible. Much shorter  
2 times should be realized in a scaled up manufacturing process where the  
3 manufacturing environment and equipment conditions are controlled to avoid  
4 water absorption.

5           After cleaning and dehydration, patterned layers are ready for  
6 alignment and lamination. In separate experiments, it was discovered that KJ and  
7 EKJ films adhere to many metal surfaces during pressurized heat-sealing in a hot  
8 press. Lamination therefore makes use of a platen separator. A high-temperature  
9 separator material is necessary to prevent the outside layers, e.g., layers 22a and  
10 22d in FIG. 1, from bonding to the platens of the hot press. Duofoil® (JJA, Inc.)  
11 was found suitable for use as a separator plate. Kapton KJ and EKJ films did not  
12 permanently adhere to Duofoil® after exposure to 300°C and 1.4MPa pressure.  
13 The platen separator should be cleaned (step 68) to avoid contamination of the  
14 polyimide. In experiments, the Duofoil® platen separator was cleaned with  
15 isopropanol. Placement of the polyimide layers on the platen separator (step 70)  
16 should be conducted with sufficient heat to avoid condensation on the layers. In  
17 experiments, an initial alignment of polyimide layers on Duofoil® sheets  
18 positioned on a flat hotplate at a constant temperature of 50-55°C staved off  
19 condensation. The process is completed with placement of a second platen  
20 separator on top of the stack. Lamination is then conducted in a vacuum hot  
21 process.

22           In experiments, a second Duofoil® plate was positioned on the four  
23 aligned polyimide layers, and the entire stack was sandwiched between two 160  
24 mm x 160 mm square aluminum plates, 25 mm thick. The aluminum block was  
25 then positioned on center in a modified Carver vacuum hot press at a standby  
26 temperature of 200°C. FIG. 4 shows the time, temperature, and applied pressure  
27 profile found to optimally bond the layers together. A pressure of 0.1 KPa was  
28 achieved in the press chamber and the press temperature was ramped to 300°C at a  
29 rate of 2°C/min. Once 300°C was reached, the hydraulic jack was used to apply a

1 pressure of approximately 1 MPa for 25 minutes. Some pressure relaxation occurs  
2 during lamination, and no controls were initiated to maintain a constant load.  
3 After the 25 minutes had elapsed, the load was disengaged and the aluminum  
4 block was removed.

5 A cooling of the laminated heat compressor (step 72) preferably  
6 includes an inversion of the structure after removal from the vacuum process. In  
7 the experiments, the aluminum blocks were removed, flipped over, placed on a flat  
8 cast iron base, and allowed to cool to room temperature over a period of two  
9 hours. Rotation of the blocks switched the orientation of the films contained  
10 within the stack, thus reversing any previously acquired sagging in the header and  
11 channel layers during the initial phase of the cool-down process. The block cools  
12 via conduction to the cast iron base or by natural convection to the surrounding  
13 air. As such, the aluminum blocks provided the thermal mass which self-  
14 controlled the cooling process.

15 Several different uniformly bonded (no interlaminar voids or  
16 bubbles), functional 100 mm x 100 mm footprint, prototype heat exchangers  
17 according to the FIGs. 1 and 2 embodiment were fabricated. The description of  
18 prototypes is included here only as an example, and the invention is not limited to  
19 the materials, dimensions or geometry of the prototypes. Empirical studies of each  
20 implemented design iteration yielded various critical fabrication parameters.  
21 During the lamination process, excessive thermoplastic flow of material in layers  
22 adjacent (above or below) to a local internal geometry can easily occlude both  
23 channels and manifolds which have micron scaled dimensions. Therefore, the  
24 most critical design parameter underlying the four-layer lamination methodology  
25 for creation of internal geometries was a material dependent, maximum allowable  
26 membrane span. For EKJ films, membrane spans up to 2 mm are allowed because  
27 of the presence of a stiff Kapton® E core with a higher apparent glass transition  
28 temperature. The maximum membrane span of KJ films are considerably less,  
29 probably closer to 500  $\mu\text{m}$ .

1           In the fabrication of experimental prototypes, channel dimensions  
2 were targeted at 75  $\mu\text{m}$  high x 800  $\mu\text{m}$  wide. However, some compression of  
3 these dimensions was noticed subsequent to lamination, resulting in approximate  
4 channel dimensions of 70  $\mu\text{m}$  x 750  $\mu\text{m}$ . Over numerous cross-sections, no  
5 discernable interface existed between the internal KJ layers (2 & 3) after bonding,  
6 direct evidence of diffuse, thermoplastic polymer welding. Moreover, plastic flow  
7 of these layers was observed in the narrowing channel width, or widening of the  
8 channel separators, towards the bottom of the channel. In qualitative strength  
9 tests, KJ/KJ welded interfaces demonstrated the highest observed bond strengths.  
10 However, because of the aforementioned sagging criterion, an all-KJ, four layer  
11 proved unfeasible.

12           Accordingly, the sequencing of EKJ and KJ films within the  
13 laminate mesoscopic heat exchanger is not an arbitrary design parameter. From  
14 this, the invention should be carried out with outer layers having a modulus and  
15 glass transition temperature to withstand lamination with thermoplastic flow and  
16 inner layers that permit limited thermoplastic flow that maintains microchannel  
17 shape during lamination. Channel dimensions can be selected depending on the  
18 application. Thinner channels than those tested in the experimental prototypes can  
19 be used if shorter channel lengths are employed, and vice versa. Moreover, the  
20 span width can be adjusted with respect to the cap layer thickness to determine  
21 how much sagging is desired. In fact, under pressure, the channel height  
22 effectively becomes larger due to expansion of the cap layer, which permits a  
23 higher flow rate. This phenomenon helps to self-regulate the pressure drop in the  
24 channels and is a benefit of the invention.

25           The fabrication method of the invention, such as the preferred  
26 method of FIG. 3, will lend itself into a mass production conducted, for example,  
27 on a moving web machine. Each layer is a separate feed into the web, with a  
28 cutting and patterning station to make its pattern. Conditions are maintained to  
29 laminate the layers after patterning while moving on the moving web.

1           While specific embodiments of the present invention have been  
2 shown and described, it should be understood that other modifications,  
3 substitutions and alternatives are apparent to one of ordinary skill in the art. Such  
4 modifications, substitutions and alternatives can be made without departing from  
5 the spirit and scope of the invention, which should be determined from the  
6 appended claims.

7           Various features of the invention are set forth in the appended  
8 claims.